# A model for confined vortex rings with elliptical core vorticity distribution

Ionut Danaila<sup>1</sup>, Felix Kaplanski<sup>2,3</sup>, and Sergei S Sazhin<sup>3</sup>

<sup>1</sup>Laboratoire de Mathématiques Raphaël Salem, Université de Rouen Normandie, F-76801 Saint-Étienne-du-Rouvray, France

<sup>2</sup>Tallinn University of Technology, Ehitajate tee 5, Tallinn 12616, Estonia
 <sup>3</sup>Sir Harry Ricardo Laboratories, Advanced Engineering Centre, School of Computing, Engineering and Mathematics, University of Brighton, Brighton BN2 4GJ, U.K.

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We present a new model for an axisymmetric vortex ring confined in a tube. The model takes into account the elliptical (elongated) shape of the vortex ring core and thus extends our previous model [Danaila, Kaplanski and Sazhin, J. of Fluid Mechanics, 774, 2015] derived for vortex rings with quasi-circular cores. The new model offers a more accurate description of the deformation of the vortex ring core, induced by the lateral wall, and a better approximation of the translational velocity of the vortex ring, compared with the previous model. The main ingredients of the model are the following: the description of the vorticity distribution in the vortex ring is based on the previous model of unconfined elliptical-core vortex rings [Kaplanski, Fukumoto and Rudi, Physics of Fluids, 24, 2012]; Brasseur's approach [Brasseur, PhD Thesis, 1979] is then applied to derive a wall-induced correction for the Stokes stream function of the confined vortex ring flow. We derive closed formulae for the flow stream function and vorticity distribution. An asymptotic expression for the long time evolution of the drift velocity of the vortex ring as a function of the ellipticity parameter is also derived. The predictions of the model are shown to be in agreement with direct numerical simulations of confined vortex rings generated by a piston-cylinder mechanism. The predictions of the model support the recently suggested heuristic relation [Krieg and Mohseni, J. Fluid Engineering, 135, 2013] between the energy and circulation of vortex rings with converging radial velocity. A new procedure for fitting experimental and numerical data with the predictions of the model is described. This opens the way for applying the model to realistic confined vortex rings in various applications including those in internal combustion engines.

#### 1. Introduction

A large amount of physical and mathematical literature has been devoted to the study of vortex rings in unbounded domains (see the reviews by Shariff & Leonard (1992), Lim & Nickels (1995), Fukumoto (2010)). The study of radially confined vortex ring flows has received far less attention, despite its importance in many practical applications. These include vortex ring-like structures in gasoline internal combustion engines (Begg et al. 2009) and transient flow fields behind arterial stenoses (Gharib et al. 2006).

To the best of the authors' knowledge, the first theoretical analysis of confined vortex rings was performed by Brasseur (1979). He assumed that the flow field in the outer region (far from the core) depends on its total circulation but not on the details of the ring's vorticity distribution. The vortex ring in his study was represented as a thin circular vortex filament (CVF) (Lamb 1932), with given initial circulation  $\Gamma_0$  and radius  $R_0$ . Then the stream function of the confined vortex ring was decomposed as a sum of an unconfined stream function for the CVF and a correction stream function induced by the tube (Brasseur (1986)). Danaila et al. (2015) applied Brasseur's method using not the CVF but a more realistic vorticity distribution in the vortex ring based on the unconfined viscous vortex ring model derived by Kaplanski & Rudi (1999, 2005). The wall-induced correction of the stream function was derived analytically, giving closed analytical formulas for the vorticity and stream function of a confined vortex ring. Using the power-law assumption for the time variation of the viscous length of the vortex ring, the time variations of the main integral characteristics such as circulation, kinetic energy and translational velocity were obtained. The predictions of this model were shown to be in agreement with available experimental data (Stewart et al. 2012) and results of direct numerical simulations.

The confined vortex ring model suggested by Danaila et al. (2015) is based on the assumption that the vorticity distribution of the vortex ring is quasi-isotropic Gaussian. This assumption was inferred from the unconfined vortex ring model suggested by Kaplanski & Rudi (1999, 2005). This model was analytically derived as a first-order solution of the linearised axisymmetric Navier-Stokes equation. The measurements of unconfined or confined vortex ring vorticity, however, show that the vorticity field of the vortex ring deforms during its development (Fukumoto & Moffatt 2000); at later stages it becomes elongated due to the Reynolds number effects (Weigand & Gharib (1997); Danaila & Helie (2008)). This deformation is accompanied by the modification of the vorticity distribution function and leads to changes in vortex ring integral characteristics. For example, the compression of the vorticity during the formation stage leads to an increase in both ring translational velocity (U) and the downstream distance at which the separation of the ring from the trailing jet occurs. At the decay stage, this leads to an enhanced reduction of the values of U. Also, the deformation of the vortex ring core has been observed in the process of the ring interaction with other vortex structures, e.g. vortex ring interaction with a columnar vortex (Candon & Marshall 2012). It was shown that taking into account the effects of ellipticity leads to better predictions of the translational velocity of the vortex ring, when compared to experimentally observed rings (Weigand & Gharib 1997) or the predictions of previously developed vortex ring models (Fukumoto & Moffatt 2008).

For a vortex ring confined in a tube, the elongated shape of the vortex core is accentuated by the interaction with the lateral wall (Danaila *et al.* 2015). Also, the tube wall induces the deformation of the streamlines of the flow, without significant changing of the vorticity field. Based on these observations, we have developed a new model for a confined vortex ring, taking into account the elliptic shape of the vortex core. As in the

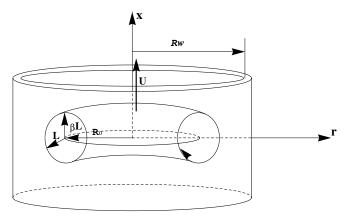


FIGURE 1. Schematic of a vortex ring in a tube. The elliptical core is described using two length scales: L in the radial direction and  $\beta L$  in the streamwise direction.

model suggested by Kaplanski et al. (2012), the ellipticity of the vorticity distribution in the vortex ring is taken into account. As in our earlier model (Danaila et al. 2015) Brasseur's method is used to take into account the stream function correction induced by the wall. In the new model, the elliptical distribution of vorticity is postulated, while other properties, such as circulation  $\Gamma$ , kinetic energy E, translational velocity U and stream function  $\Psi$  are calculated. The main motivation for using a more realistic elliptical description of the vorticity distribution in the vortex rings is to render the model useful for practical applications requiring an accurate but simple model for describing the geometry of confined vortex rings (e. g. vortex rings in gasoline internal combustion engines (Begg et al. (2009); Sazhin (2014)) or for reconstructing the velocity field around a vortex ring when only incomplete measurements are available (see Zhang & Danaila (2012); Danaila & Protas (2015)).

The paper is organized as follows. Previously developed analytical vortex ring models taking into account the effects of the radial confinement and ellipticity of the vortex ring core are summarised in §2. In §3 a new model for the confined elliptical-core vortex ring is described. In §4 direct numerical simulations of axisymmetric vortex rings are used to assess the applicability of the new model. The predictions of the new model are compared with the heuristic relation between vortex ring energy and circulation suggested by Krieg & Mohseni (2013) in §5. The main results of the paper are summarised in §6.

# 2. Background

Danaila et al. (2015) considered an axisymmetric vortex ring of radius  $R_0$  placed in an infinitely long rigid tube of radius  $R_w$ , moving in an incompressible fluid with translational velocity U along the axis of symmetry (x) (see figure 1). For a given time instant t, the vorticity  $\omega(r, x)$  and the Stokes stream function  $\Psi(r, x)$  are determined by the following equations (Batchelor 1988):

$$\mathcal{L}\Psi = \omega, \quad \mathcal{L} := -\frac{\partial}{\partial r} \left( \frac{1}{r} \frac{\partial}{\partial r} \right) - \frac{\partial}{\partial x} \left( \frac{1}{r} \frac{\partial}{\partial x} \right) = -\frac{1}{r} \left( \frac{\partial^2}{\partial r^2} + \frac{\partial^2}{\partial x^2} - \frac{1}{r} \frac{\partial}{\partial r} \right), \quad (2.1)$$

where x, r are the axes of a cylindrical coordinate system. We consider the following boundary conditions; at the axis of symmetry:

$$\omega(0,x) = \Psi(0,x) = 0, \text{ for } r = 0,$$
 (2.2)

vanishing vorticity and stream function in the far field:

$$\omega, \Psi \to 0 \quad \text{when} \quad \left(x^2 + r^2\right)^{1/2} \to \infty,$$
 (2.3)

and no flow through the tube wall:

$$\omega \to 0, \quad \frac{1}{r} \frac{\partial \Psi}{\partial x} = 0, \quad \text{for} \quad r = R_w.$$
 (2.4)

Brasseur (1979) represented the vortex ring as a circular vortex filament (CVF) and derived a correction of the stream function of the flow to satisfy the boundary condition (2.4) at the wall. Danaila et al. (2015) used an approach similar to that used by Brasseur (1979), but replaced CVF with the more realistic vortex ring model derived by Kaplanski & Rudi (1999, 2005). This was possible since the vortex filament does not include any description of the vortex geometry. In what follows we summarise the main formulae used in the model developed by Danaila et al. (2015). This model is hereafter referred to as the vortex ring model with a circular core (VRC) and subscript VRC will be used to identify the parameters obtained in this model.

Danaila et al. (2015) scaled physical variables as:

$$x_1 = \frac{x - X_0}{R_0}, \quad r_1 = \frac{r}{R_0}, \quad \omega^* = \frac{\omega R_0^2}{\Gamma_0}, \quad \Psi^* = \frac{\Psi}{\Gamma_0 R_0},$$
 (2.5)

where  $R_0$  is the radius of the vortex ring,  $\Gamma_0$  its initial circulation and  $X_0$  the streamwise position of the vortex centre. In what follows, the star superscripts are dropped and the vorticity and the stream function are considered as dimensionless. The vortex ring model derived by Kaplanski & Rudi (1999, 2005) is described by parameter  $\theta_c$ :

$$\theta_c = \frac{R_0}{L_c},\tag{2.6}$$

where  $L_c$  is the diffusivity length (see figure 1), which is assumed to vary in time following a power-law for laminar vortex rings:

$$\theta_c = \frac{R_0}{\sqrt{2\nu t}},\tag{2.7}$$

where  $\nu$  is the fluid kinematic viscosity. The generalisation of Eq (2.7) for turbulent vortex rings was discussed by Kaplanski *et al.* (2009). The Reynolds number of the flow is introduced as  $Re = \Gamma_0/\nu$  and the parameter  $\varepsilon = R_0/R_w < 1$  quantifies the confinement of the vortex ring. The wall is located at  $r_{1w} = 1/\varepsilon$ .

Following Brasseur's approach, stream function  $\Psi^w$  describing the flow generated by a viscous vortex ring in a tube is presented as the sum of the stream function of the unbounded vortex ring  $\Psi$  and the wall-induced correction  $-\Psi^0$ . In the VRC model the stream function of the flow was presented as:

$$\Psi_{\text{VRC}}^{w}(x_1, r_1) = \Psi_{\text{VRC}}(x_1, r_1) - \Psi_{\text{VRC}}^{0}(x_1, r_1), \tag{2.8}$$

where

$$\Psi_{\text{VRC}} = \frac{r_1 \theta_c}{4} \int_0^\infty \left[ \exp(x_1 \theta_c \mu) \operatorname{erfc}(\frac{\mu + x_1 \theta_c}{\sqrt{2}}) + \exp(-x_1 \theta_c \mu) \operatorname{erfc}(\frac{\mu - x_1 \theta_c}{\sqrt{2}}) \right] J_1(r_1 \theta_c \mu) J_1(\theta_c \mu) d\mu.$$
(2.9)

Stream function  $\Psi_{VRC}$  corresponds to the vorticity distribution predicted by the unbounded vortex ring model derived by Kaplanski & Rudi (1999, 2005):

$$\omega_{\text{VRC}} = \frac{\theta_c^3}{\sqrt{2\pi}} \exp\left(-\frac{(r_1^2 + x_1^2 + 1)\theta_c^2}{2}\right) I_1\left(r_1\theta_c^2\right). \tag{2.10}$$

In (2.9) and (2.10),  $J_1$  is the Bessel function of the first kind,  $I_1$  the modified Bessel function of the first kind and erfc the complementary error function

$$\operatorname{erfc}(x) = \frac{2}{\sqrt{\pi}} \int_{x}^{\infty} \exp(-t^{2}) dt. \tag{2.11}$$

Using series expansion in (2.9) in terms of  $z_1 = \theta_c \sqrt{x_1^2 + r_1^2} \to \infty$ , it was shown that the limit of the stream function (2.9) at a large distance from the ring's core is identical to that found by Brasseur (1979) for the circular vortex filament. The stream function correction  $\Psi_{\text{VRC}}^0$  induced by the presence of the tube was then found as:

$$\Psi_{\text{VRC}}^{0} = \frac{r_1}{\pi} \int_0^\infty \frac{K_1(\mu/\varepsilon)}{I_1(\mu/\varepsilon)} I_1(\mu) I_1(r_1\mu) \cos(x_1\mu) d\mu, \qquad (2.12)$$

where  $K_1$  is the modified Bessel function of the second kind.

The stream function (2.8) along with the vorticity distribution (2.10) represents an approximate solution to problem (2.1) with boundary conditions (2.2)-(2.4). Using the power-law assumption (2.7), the time-dependent integral characteristics of the confined vortex ring flow based on the VRC model were predicted. When compared with those obtained by direct numerical simulations, a good agreement was obtained. The largest difference between analytical and numerical results was observed for relatively high Reynolds numbers, when the quasi-circular shape of the vortex core resulting from the vorticity distribution (2.10) turned out not to be very accurate. It is anticipated that the accuracy of a more advanced model taking into account realistic elliptical shapes of vortex ring cores, described in the next section, will improve the accuracy of the Danaila et al. (2015) model.

The confined vortex ring model, taking into account the ellipticity of the vortex ring core, described later in Section §3, is based on the model suggested by Kaplanski *et al.* (2012) for an unbounded viscous vortex ring with an elliptical cross section.

This model, hereafter referred to as the VRE model, is based on imposing weak perturbations on the first-order solution (2.10) for the vorticity in order to find an approximation of the non-linear solution to the Navier-Stokes equations for high Reynolds numbers. Parameter  $\beta$  (see figure 1) allowed Kaplanski *et al.* (2012) to take into account the deformation of the vortex ring core along the streamwise direction. Using scaling (2.5), the non-dimensional vorticity for the VRE model was expressed as:

$$\omega_{\text{VRE}} = \frac{\theta_e^3}{\beta \sqrt{2\pi}} \exp\left(-\frac{(r_1^2 + (x_1/\beta)^2 + 1)\theta_e^2}{2}\right) I_1\left(r_1\theta_e^2\right), \tag{2.13}$$

where

$$\theta_e = \frac{R_0}{L_e} = \lambda \theta_c \Longrightarrow L_e = \frac{L_c}{\lambda},$$
(2.14)

 $L_e$  is the new viscous length scale.

Parameters  $\beta > 0$  and  $\lambda > 0$  describe the elongation or compression along axes x and r, respectively, when compared to the VRC model for the vortex ring with circular cross section. Furthermore, two small parameters  $\epsilon_0$  and  $\gamma^0$  were introduced, such that

$$\beta = 1 + \epsilon_0, \quad \lambda = 1 + \gamma^0 \tag{2.15}$$

where  $0 \le \epsilon_0 < 1$  and  $0 \le \gamma^0 < 1$ .

The stream function corresponding to vorticity distribution (2.13) was derived in the form:

$$\Psi_{\text{VRE}} = \frac{r_1 \theta_e}{4} \int_0^\infty \exp\left((\beta^2 - 1) \frac{\mu^2}{2}\right) \left[\exp(\mu x_1 \theta_e) \operatorname{erfc}\left(\frac{\mu \beta + x_1 \theta_e / \beta}{\sqrt{2}}\right)\right]$$

$$+\exp(-\mu x_1 \theta_e)\operatorname{erfc}\left(\frac{\mu\beta - x_1 \theta_e/\beta}{\sqrt{2}}\right) J_1(\theta_e \mu) J_1(r_1 \theta_e \mu) d\mu. \quad (2.16)$$

After the change of variables  $\mu\theta_e \to \mu$  in the integrand, at large distances  $z = \theta_e \sqrt{x_1^2 + r_1^2} \to \infty$  the stream function (2.16) was approximated by:

$$\Psi_{\text{VRE}} \approx \Psi_{\text{VRE}}^a = \frac{r_1}{2} \int_0^\infty \exp\left((\beta^2 - 1) \frac{\mu^2}{2\theta_e^2}\right) \exp(-|x_1|\mu) J_1(\mu) J_1(r_1\mu) d\mu.$$
(2.17)

In the next section, the previously developed models, described above, will be combined to investigate the radially confined vortex rings with elliptical cores.

## 3. Radially confined vortex rings with elliptical cores

Brasseur's methodology is applied to derive the stream function of the flow generated in a tube by the viscous vortex ring with vorticity distribution (2.13). A new scaling, based on  $R_w$  as length scale, is introduced:

$$\tilde{x} = \frac{x}{R_w} = \varepsilon x_1, \quad \tilde{r} = \frac{r}{R_w} = \varepsilon r_1,$$
(3.1)

where  $\varepsilon = R_0/R_w$  was defined earlier. The total stream function of the flow is decomposed as:

$$\Psi_{\text{VRE}}^w = \Psi_{\text{VRE}} - \Psi_{\text{VRE}}^0, \tag{3.2}$$

where the correction  $\Psi_{\text{VRE}}^0$  satisfies the equation  $\mathcal{L}\Psi_{\text{VRE}}^0 = 0$ , with operator  $\mathcal{L}$  defined in (2.1), and the following boundary conditions at the wall (cf. (2.4)):

$$\frac{1}{\tilde{r}} \frac{\partial \Psi_{\text{VRE}}^0}{\partial \tilde{r}} = \frac{1}{\tilde{r}} \frac{\partial \Psi_{\text{VRE}}}{\partial \tilde{r}}, \text{ at } \tilde{r} = 1.$$
 (3.3)

An equivalent formulation is obtained by introducing the flow potential  $\Phi$  linked with  $\Psi$  by the following equations, giving the two components of the velocity vector:

$$v_r = \frac{\partial \Phi}{\partial r} = -\frac{1}{r} \frac{\partial \Psi}{\partial x}, \quad v_x = \frac{\partial \Phi}{\partial x} = \frac{1}{r} \frac{\partial \Psi}{\partial r}.$$
 (3.4)

This allows us to present an alternative form of (3.2):

$$\Phi_{\text{VRE}}^w = \Phi_{\text{VRE}} - \Phi_{\text{VRE}}^0, \tag{3.5}$$

with the correction potential satisfying the Laplace equation:

$$\nabla^2 \Phi_{\text{VRE}}^0 = 0, \tag{3.6}$$

and the following boundary condition at the wall:

$$\frac{\partial \Phi_{\text{VRE}}^0}{\partial \tilde{r}} = \frac{\partial \Phi_{\text{VRE}}}{\partial \tilde{r}}, \text{ at } \tilde{r} = 1.$$
 (3.7)

To derive an approximation of  $\Phi_{\text{VRE}}^0$  we assume that approximation (2.17) is valid in the vicinity of the wall. The non-dimensional potential (scaled by  $\Gamma_0$ ) corresponding to the stream function (2.17) is obtained in the form:

$$\Phi_{\text{VRE}}^{a} = -\frac{1}{2} \int_{0}^{\infty} \exp\left((\beta^{2} - 1) \frac{\mu^{2}}{2\theta_{s}^{2}}\right) \exp(-|\tilde{x_{1}}|\mu) J_{1}(\mu) J_{0}(r_{1}\mu) d\mu. \tag{3.8}$$

This equation is rewritten in  $(\tilde{r}, \tilde{x})$  coordinates as:

$$\Phi_{\text{VRE}}^{a} = -\frac{\varepsilon}{2} \int_{0}^{\infty} \exp\left( (\beta^{2} - 1) \frac{\varepsilon^{2} \mu^{2}}{2\theta_{e}^{2}} \right) \exp(-|\tilde{x}|\mu) J_{1}(\varepsilon \mu) J_{0}(\tilde{r}\mu) d\mu.$$
 (3.9)

Since  $\beta = 1 + \epsilon_0$  (see (2.15)), we can use the following power series expansion in terms of the small parameter  $\epsilon_0$ :

$$\exp\left((\beta^{2} - 1)\frac{\varepsilon^{2}\mu^{2}}{2\theta_{e}^{2}}\right) = \exp\left((2\epsilon_{0} + \epsilon_{0}^{2})\frac{\chi^{2}\mu^{2}}{2}\right) = 1 + \epsilon_{0}\chi^{2}\mu^{2} + \frac{1}{2}\epsilon_{0}^{2}\chi^{2}\mu^{2}\left(1 + \chi^{2}\mu^{2}\right) + ...,$$
(3.10)

where

$$\chi = \frac{\varepsilon}{\theta_e} = \frac{L_e}{R_m} \tag{3.11}$$

represents the ratio between the viscous length scale and the radius of the tube ( $\chi \ll 1$ ). Using the expansion

$$J_1(\varepsilon\mu) = \frac{\varepsilon\mu}{2} - \frac{(\varepsilon\mu)^3}{16} + \frac{(\varepsilon\mu)^5}{284} + \dots, \tag{3.12}$$

and remembering that

$$\frac{\partial^n}{\partial x^n} \exp(-\mu x) = (-\mu)^n \exp(-\mu x) \tag{3.13}$$

potential (3.9) can be expanded as:

$$\Phi_{\text{VRE}}^{a} = \frac{\varepsilon}{2} \left( \frac{\varepsilon}{2} \frac{\partial S}{\partial \tilde{x}} - \frac{\varepsilon^{3}}{16} \frac{\partial^{3} S}{\partial \tilde{x}^{3}} + \frac{\varepsilon^{5}}{284} \frac{\partial^{5} S}{\partial \tilde{x}^{5}} - \dots \right), \tag{3.14}$$

where

$$S = \int_0^\infty \left( 1 + \epsilon_0 \chi^2 \mu^2 + \frac{1}{2} \epsilon_0^2 \chi^2 \mu^2 \left( 1 + \chi^2 \mu^2 \right) \right) \exp(-|\tilde{x}|\mu) J_0(\tilde{r}\mu) d\mu.$$
 (3.15)

Note that we kept an approximation of the order of  $o(\epsilon_0^2)$ . This approach can be extended to higher order approximations in  $\epsilon_0$ . From (3.14) and (3.15) it can be seen that the behaviour of potential  $\Phi_{\text{VRE}}^a$  at large distances is controlled by S; for  $\epsilon_0 = 0$  the first harmonic of (3.14) represents a point dipole (although, in fact, a similar dependence of this potential on  $\tilde{x}$  and  $\tilde{r}$  takes place for any  $\epsilon_0$ ).

The basic idea behind Brasseur's method is that expansion (3.14) can be applied to the total flow field  $\Phi^w_{\text{VRE}}$ . In this case, function S is replaced by function  $Q = S - Q_1$  which satisfies the boundary condition of no flow through the tube wall  $(\partial Q/\partial \tilde{r} = 0)$  at  $\tilde{r} = 1$ . Following decomposition (3.5), function  $Q_1$  is finally used to derive the correction potential as:

$$\Phi_{\text{VRE}}^{0} = \frac{\varepsilon}{2} \left( \frac{\varepsilon}{2} \frac{\partial Q_1}{\partial \tilde{x}} - \frac{\varepsilon^3}{16} \frac{\partial^3 Q_1}{\partial \tilde{x}^3} + \frac{\varepsilon^5}{284} \frac{\partial^5 Q_1}{\partial \tilde{x}^5} - \dots \right). \tag{3.16}$$

Let us now rewrite (3.15) as:

$$S = D_1 + \left(\epsilon_0 \chi^2 + \frac{1}{2} \epsilon_0^2 \chi^2\right) D_2 + \frac{1}{2} \epsilon_0^2 \chi^4 D_3, \tag{3.17}$$

where

$$D_{1} = \int_{0}^{\infty} \exp(-|\tilde{x}|\mu) J_{0}(\tilde{r}\mu) d\mu = \frac{1}{\sqrt{(\tilde{x}^{2} + \tilde{r}^{2})}} = \frac{2}{\pi} \int_{0}^{\infty} K_{0}(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu, \qquad (3.18)$$

$$D_2 = \int_0^\infty \mu^2 \exp(-|\tilde{x}|\mu) J_0(\tilde{r}\mu) d\mu = \frac{(-\tilde{r}^2 + 2\tilde{x}^2)}{(\tilde{x}^2 + \tilde{r}^2)^{5/2}} = -\frac{2}{\pi} \int_0^\infty \mu^2 K_0(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu, \quad (3.19)$$

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$$D_{3} = \int_{0}^{\infty} \mu^{4} \exp(-|\tilde{x}|\mu) J_{0}(\tilde{r}\mu) d\mu = \frac{3(3\tilde{r}^{4} - 24\tilde{r}^{2}\tilde{x}^{2} + 8\tilde{x}^{4})}{(\tilde{x}^{2} + \tilde{r}^{2})^{9/2}} = \frac{2}{\pi} \int_{0}^{\infty} \mu^{4} K_{0}(\tilde{r}\mu) \cos(\mu\tilde{x}) d\mu.$$
(3.20)

Function Q is sought in the following form:

$$Q = \left[ D_1 + \int_0^\infty f_1(\mu) \mathbf{I}_0(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu \right]$$

$$+ \left[ \left( \epsilon_0 \chi^2 + \frac{1}{2} \epsilon_0^2 \chi^2 \right) D_2 + \int_0^\infty f_2(\mu) \mu^2 \mathbf{I}_0(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu \right]$$

$$+ \left[ \frac{1}{2} \epsilon_0^2 \chi^4 D_3 + \int_0^\infty f_3(\mu) \mu^4 \mathbf{I}_0(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu \right].$$
(3.21)

Remembering the definitions of integrals  $D_1$ ,  $D_2$  and  $D_3$ , this expression can be rewritten as:

$$Q = \int_0^\infty \left[ \frac{2}{\pi} K_0(\tilde{r}\mu) + f_1(\mu) I_0(\tilde{r}\mu) \right] \cos(\mu \tilde{x}) d\mu$$

$$+ \int_0^\infty \left[ \left( \epsilon_0 \chi^2 + \frac{1}{2} \epsilon_0^2 \chi^2 \right) \left( -\frac{2}{\pi} \right) K_0(\tilde{r}\mu) + f_2(\mu) I_0(\tilde{r}\mu) \right] \mu^2 \cos(\mu \tilde{x}) d\mu$$

$$+ \int_0^\infty \left[ \frac{1}{2} \epsilon_0^2 \chi^4 \left( \frac{2}{\pi} \right) K_0(\tilde{r}\mu) + f_3(\mu) I_0(\tilde{r}\mu) \right] \mu^4 \cos(\mu \tilde{x}) d\mu. \tag{3.22}$$

The boundary condition  $\partial Q/\partial \tilde{r} = 0$  at  $\tilde{r} = 1$  requires that

$$f_1(\mu) = \frac{2}{\pi} \frac{K_1(\mu)}{I_1(\mu)},$$
 (3.23)

$$f_2(\mu) = \left(\epsilon_0 \chi^2 + \frac{1}{2} \epsilon_0^2 \chi^2\right) \left(-\frac{2}{\pi}\right) \frac{K_1(\mu)}{I_1(\mu)},$$
 (3.24)

$$f_3(\mu) = \frac{1}{2} \epsilon_0^2 \chi^4 \left(\frac{2}{\pi}\right) \frac{K_1(\mu)}{I_1(\mu)}.$$
 (3.25)

Using these expressions for  $f_1$ ,  $f_2$  and  $f_3$  and (3.21), we obtain the new expression for function  $Q = S - Q_1$ , with:

$$Q_{1} = -\frac{2}{\pi} \int_{0}^{\infty} \left[ 1 - \left( \epsilon_{0} \chi^{2} + \frac{1}{2} \epsilon_{0}^{2} \chi^{2} \right) \mu^{2} + \frac{1}{2} \epsilon_{0}^{2} \chi^{4} \mu^{4} \right] \frac{K_{1}(\mu)}{I_{1}(\mu)} I_{0}(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu.$$
 (3.26)

Substituting this expression for  $Q_1$  into (3.16), we obtain, after differentiation, the following expression:

$$\Phi_{\text{VRE}}^{0} = \frac{\varepsilon}{\pi} \int_{0}^{\infty} \left[ \frac{\varepsilon \mu}{2} + \frac{\varepsilon^{3} \mu^{3}}{16} + \frac{\varepsilon^{5} \mu^{5}}{284} \right] \times \left[ 1 - \left( \epsilon_{0} + \frac{1}{2} \epsilon_{0}^{2} \right) \chi^{2} \mu^{2} + \frac{1}{2} \epsilon_{0}^{2} \chi^{4} \mu^{4} \right] \frac{K_{1}(\mu)}{I_{1}(\mu)} I_{0}(\tilde{r}\mu) \sin(\mu \tilde{x}) d\mu. \tag{3.27}$$

Keeping in mind that the expression in the first brackets represents the expansion of  $I_1(\varepsilon\mu)$ , we can present the potential field induced by the presence of the tube as:

$$\Phi_{\text{VRE}}^0 = \frac{\varepsilon}{\pi} \int_0^\infty \left[ 1 - \left( \epsilon_0 + \frac{1}{2} \epsilon_0^2 \right) \chi^2 \mu^2 + \frac{1}{2} \epsilon_0^2 \chi^4 \mu^4 \right] \frac{K_1(\mu)}{I_1(\mu)} I_0(\tilde{r}\mu) I_1(\varepsilon\mu) \sin(\mu \tilde{x}) d\mu. \tag{3.28}$$

Note that the terms in the square brackets represent the first terms in the series expansion of  $\exp(-(\beta^2-1)\frac{\chi^2\mu^2}{2})$ . If we go back to (3.10) and keep the higher order

terms, we obtain in (3.17) only integrals depending on even powers of  $\mu$ . These integrals have the following structure:

$$D_{2p} = \int_0^\infty \mu^{2p} \exp(-|\tilde{x}|\mu) J_0(\tilde{r}\mu) d\mu.$$
 (3.29)

It can be proven that (see Appendix A):

$$D_{2p} = \frac{2(-1)^p}{\pi} \int_0^\infty \mu^{2p} K_0(\tilde{r}\mu) \cos(\mu \tilde{x}) d\mu.$$
 (3.30)

Using (3.30) to take into account high order terms in the expansion (3.10), it can be finally shown that:

$$\Phi_{\text{VRE}}^{0} = \frac{\varepsilon}{\pi} \int_{0}^{\infty} \exp\left(-(\beta^2 - 1)\frac{\chi^2 \mu^2}{2}\right) \frac{K_1(\mu)}{I_1(\mu)} I_0(\tilde{r}\mu) I_1(\varepsilon\mu) \sin(\mu \tilde{x}) d\mu. \tag{3.31}$$

The corresponding stream function is easily derived from its definition (3.4):

$$\Psi_{\text{VRE}}^{0} = \frac{\varepsilon \tilde{r}}{\pi} \int_{0}^{\infty} \exp\left(-(\beta^{2} - 1)\frac{\chi^{2} \mu^{2}}{2}\right) \frac{K_{1}(\mu)}{I_{1}(\mu)} I_{1}(\tilde{r}\mu) I_{1}(\varepsilon\mu) \cos(\mu \tilde{x}) d\mu. \tag{3.32}$$

Remembering (3.1), expression (3.32) can be rewritten in  $(r_1, x_1)$  coordinates as:

$$\Psi_{\text{VRE}}^{0} = \frac{r_{1}}{\pi} \int_{0}^{\infty} \exp\left(-(\beta^{2} - 1)\frac{\mu^{2}}{2\theta_{e}^{2}}\right) \frac{K_{1}(\mu/\varepsilon)}{I_{1}(\mu/\varepsilon)} I_{1}(r_{1}\mu) I_{1}(\mu) \cos(\mu x_{1}) d\mu. \tag{3.33}$$

The same expression but in (r, x) coordinates can be presented as:

$$\Psi_{\text{VRE}}^{0} = \frac{r}{\pi} \int_{0}^{\infty} \exp\left(-(\beta^{2} - 1)\frac{R_{0}^{2}\mu^{2}}{2\theta_{e}^{2}}\right) \frac{K_{1}(R_{w}\mu)}{I_{1}(R_{w}\mu)} I_{1}(r\mu) I_{1}(R_{0}\mu) \cos(\mu x) d\mu. \tag{3.34}$$

The total stream function of the flow generated by a viscous vortex ring with elliptical core is inferred from (3.2), where  $\Psi_{\text{VRE}}$  is given by (2.16), and  $\Psi_{\text{VRE}}^0$  given by (3.33). Note that for  $\epsilon_0 = 0$ , (2.12) derived for the VRC model is recovered.

#### 3.1. Estimations of the ellipticity parameters

Firstly, we focus on an estimation of the ellipticity parameter  $\beta$  in a long time limit,  $t \to \infty$  ( $\theta = R_0/\sqrt{2\nu t} \to 0$ ). To simplify the analysis, we assume that the influence of the boundary layer generated at the lateral wall is weak and focus on the asymptotic development of the translational velocity U.

In the circular VRC model, the expression for the translational velocity was obtained by Kaplanski & Rudi (1999):

$$U_{\text{VRC}} = \frac{\Gamma_0 \theta_c \sqrt{\pi}}{4\pi R_0} \left[ 3 \exp(-\frac{\theta_c^2}{2}) I_1 \left( \frac{\theta_c^2}{2} \right) + \frac{\theta_c^2}{12} {}_2F_2 \left( \left\{ \frac{3}{2}, \frac{3}{2} \right\}, \left\{ \frac{5}{2}, 3 \right\}, -\theta_c^2 \right) - \frac{3\theta_c^2}{5} {}_2F_2 \left( \left\{ \frac{3}{2}, \frac{5}{2} \right\}, \left\{ 2, \frac{7}{2} \right\}, -\theta_c^2 \right) \right].$$
 (3.35)

The expansion  $U_{\text{VRC}}$  for small  $\theta_c$  gives the well-known formula derived by Rott & Cantwell (1993) for the drift velocity of a decaying vortex ring:

$$U_{\text{vrc}}^{\text{asymp}} = 0.0037 \frac{2\sqrt{2}\Gamma_0 \pi \theta_c^3}{R_0} = 0.0037 \frac{I}{\rho(\nu t)^{3/2}},$$
(3.36)

where the initial vortex ring circulation and impulse are related by  $\Gamma_0 = M/(\pi R_0^2) = I/(\rho \pi R_0^2)$ , superscript asymp indicates the asymptotic limit.

For a vortex ring with elliptic core, Kaplanski et al. (2012) derived an expression for the translational velocity in the form:

$$U_{\text{VRE}} = \frac{\Gamma_0 \theta_e}{4R_0 \pi} \int_0^\infty \exp(-\mu^2) \left[ 6\sqrt{\pi} \beta \mu + \pi \exp(\beta^2 \mu^2) (1 - 6\beta^2 \mu^2) \operatorname{erfc}(\beta \mu) \right] J_1^2(\theta_e \mu) d\mu.$$
(3.37)

Following (2.14) and (2.15), we can express the time evolution of  $\theta_e$  by relating it to that of a circular vortex ( $\theta_c$ ) (see Kaplanski *et al.* (2012)) as:

$$\theta_e(t) = \lambda(t)\theta_c(t) = (1 + \gamma_0(\theta_c(t)/\theta_{c0}))\theta_c(t), \quad \text{for} \quad t \geqslant t_0 \ (\theta_c(t) < \theta_{c0}), \tag{3.38}$$

where

$$\gamma_0(\theta_c(t)/\theta_{c0}) = \gamma^0,$$

 $\theta_{c0}$  refers to the time instant  $t_0$  such that at  $t > t_0$  the vortex ring can be considered a thick one. The value of  $t_0$  was estimated from numerical (Danaila & Helie 2008) and experimental (Weigand & Gharib 1997) data. This value turned out to be close to the one inferred from relation  $\theta_{c0} = 4$ . Expansion (3.37) with respect to small  $\theta_c$  and  $\epsilon_0$  gives:

$$U_{\text{VRE}}^{\text{asym}} = \frac{(49 - 15\epsilon_0)}{840(2\pi)^{3/2}} \frac{2\sqrt{2}\Gamma_0 \pi \theta_c^3}{R_0} = \frac{(49 - 15\epsilon_0)}{840(2\pi)^{3/2}} \frac{I}{\rho(\nu t)^{3/2}}.$$
 (3.39)

When deriving Expression (3.39) it was assumed that  $\epsilon_0 = \text{const}$  which implies that  $\beta = \text{const}$ .

Formula (3.39) is a generalisation of the Rott & Cantwell (1993) formula (see Eq. (3.36)) to the case of vortex rings with elliptical cores (this generalisation is important for high Reynolds numbers). Eq. (3.36) is recovered when  $\epsilon_0 = 0$ . It is important to link (3.39) with the formula suggested in the experimental study by Weigand & Gharib (1997):

$$U = 0.0032 \frac{I}{\rho(\nu t)^{3/2}}. (3.40)$$

The constant in (3.40) can be recovered from formula (3.39) if the ellipticity parameter is  $\epsilon_0 = 0.4$ , i. e.  $\beta = 1.4$ . This value of  $\beta = 1.4$  is also supported by the experimental data reported by Cater et al. (2004), in which both axial and radial vorticity profiles were measured (cf. Kaplanski et al. (2012)).

Formula (3.39) shows the dependency of the drift velocity of the vortex ring on the ellipticity parameter in the asymptotic limit  $t \to \infty$  ( $\theta_c \to 0$ ).

In a short time limit,  $\theta_c$  is large and  $\theta_e$  varies with time as (Kaplanski et al. 2012):

$$\theta_e(t) = \lambda(t)\theta_c(t) = (1 + \gamma_0(\theta_{c0}/\theta_c(t)))\theta_c(t), \quad \text{for} \quad t \leqslant t_0 \ (\theta_c(t) > \theta_{c0}). \tag{3.41}$$

In this case, formula (3.37) can be approximated by the asymptotic expression derived by Fukumoto & Moffatt (2008)

$$U_{cs} = \frac{\Gamma_0}{4\pi R_0} (\log(4\sqrt{2}\theta_c) - 0.558 - \frac{1.8358}{\theta_c^2})$$
 (3.42)

by considering fixed values for  $\beta$  and  $\theta_{c0}$  and fitting the value of  $\gamma_0$ . Considering two typical values of  $\theta_{c0} = 4.5$  and 4.0, we show in figure 2 the variation of the parameter  $\gamma_0$  with respect to  $\theta_c$  for three values of the ellipticity parameter  $\beta$ . Note the slow decrease in  $\gamma_0$  with  $\theta_c$  (thus an increase in time). The values of  $\gamma_0$  shown in figure 2 are close to the one predicted by Kaplanski *et al.* (2012):  $\gamma_0 = 0.16$ .

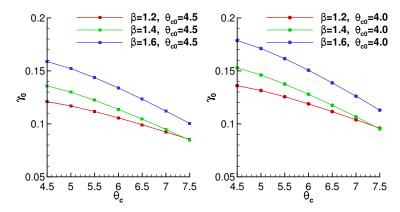


FIGURE 2. The model parameter  $\gamma_0$  for two values of  $\theta_{c0}$  and three values of the ellipticity parameter  $\beta$ .

#### 4. Predictions of the model versus direct numerical simulations

### 4.1. Physical and numerical parameters

The numerical code for solving the incompressible Navier-Stokes equations in cylindrical coordinates is described by Danaila et al. (2015) (see also Danaila & Helie (2008), Danaila et al. (2009)). The physical and numerical parameters, used in our simulations, are summarised below. The vortex ring was generated numerically by prescribing an appropriate axial velocity profile at the inlet section of the computational domain. This simulates the velocity profile produced by a vortex ring generator of piston/cylinder type. The main physical parameter of the flow is the Reynolds number based on the diameter D of the vortex generator and the mean injection velocity  $U_0$ :

$$Re_{\rm D} = \frac{U_0 D}{\nu}.\tag{4.1}$$

Using the time scale  $t_0 = D/U_0$ , the non-dimensional time used in simulations is defined as  $\tau = tU_0/D$ . We consider only short injection times, as in experiments performed by Stewart *et al.* (2012). This allows us to prescribe the inflow velocity using the model suggested by Danaila *et al.* (2009):

$$U_{\rm in}(r,\tau) = U_0 U_p(\tau) U_{\rm CL}(\tau) U_b(r,\tau), \quad 0 < \tau \leqslant \tau_{\rm off}, \tag{4.2}$$

where  $U_p(\tau)$  is the piston velocity program (normalised by its maximum value  $U_0$ ),  $U_{\text{CL}}(\tau)$  takes into account the time evolution of the centreline velocity,

$$U_{\rm CL}(\tau) = \frac{1}{1 - \frac{8}{\sqrt{\pi R e_{\rm D}}} \sqrt{\tau} + \frac{8}{R e_{\rm D}} \tau}$$
(4.3)

and  $U_b(r,\tau)$  describes the radial dependence of the profile

$$U_b(r,\tau) = \frac{1}{2} \left\{ 1 + \tanh\left[\frac{1}{4\Theta(\tau)} \left(1 - \frac{r}{R_{\rm jet}(\tau)}\right)\right] \right\},\tag{4.4}$$

$$\Theta(\tau) = \frac{\sqrt{2} - 1}{\sqrt{\pi}} B(\tau), \quad R_{\text{jet}}(\tau) = \frac{1}{2} - 0.477 B(\tau), \quad B(\tau) = \frac{2}{\sqrt{Re_{D}}} \sqrt{\tau}.$$
(4.5)

	$ au_1$	$ au_2$	$ au_{ m off}$	$L_p/D$
Case 1	1.57	2.26	2.27	1.130
Case 2	0.80	1.15	1.16	0.575

Table 1. Parameters  $\tau_1, \tau_2, \tau_{\text{off}}$  of the injection program (4.6) and piston stroke ratios  $L_p/D = \int_0^{\tau_{\text{off}}} U_p(\tau) d\tau$  used for Cases 1 and 2 of direct numerical simulations.

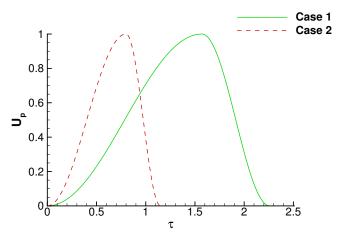


FIGURE 3. Piston velocity  $U_p(\tau)$  used for DNS run cases described in table 1. Case 1 refers to the recent experiments by Stewart *et al.* (2012).

For the piston velocity program we used the model suggested by Zhao et al. (2000):

$$U_p(\tau) = \begin{cases} \frac{1}{2} \left[ 1 - \cos\left(\pi \frac{\tau}{\tau_1}\right) \right], & \tau \leqslant \tau_1 \\ \frac{1}{2} \left[ 1 + \cos\left(\pi \frac{\tau - \tau_1}{\tau_2 - \tau_1}\right) \right], & \tau_1 < \tau \leqslant \tau_2 \approx \tau_{\text{off}}. \end{cases}$$

$$(4.6)$$

Function (4.6) gives a good approximation of the piston velocity  $U_p(\tau)$  observed by Stewart *et al.* (2012).

We consider the following two run cases described in table 1. Case 1 (see figure 3, the solid curve) corresponds to experiments by Stewart et al. (2012) and was considered by Danaila et al. (2015) to test the VRC model. It will be used to compare the two confined vortex ring models. A shorter injection program was considered for Case 2 (see figure 3, the dashed curve), giving an approximatively two times smaller piston stroke ratio  $L_p/D$ . Case 2 is expected to produce vortex rings with less elongated cores. Note that these stroke ratios are lower than the so-called formation number of the vortex ring, reported to be in the range 3.6 to 4.5 (see Gharib et al. (1998); Krueger & Gharib (2003); Dabiri & Gharib (2005)). This implies that all the vorticity produced by the vortex generator is expected to be engulfed by the vortex ring.

In all simulations the Reynolds number was set to  $Re_D = 3400$ , as in Danaila et al. (2015), since this relatively high value proved challenging in testing the VRC model. The

length of the computational domain was taken as  $L_d=8$ , which allowed us to avoid the vortex ring coming close to the downstream boundary. The grid was uniform in both axial and radial directions with a refined grid size  $\delta r=\delta x=0.01$  which ensures grid convergence of the results. Tests with a stretched grid in the radial direction, such that at least 30 grid points are clustered in the vorticity layer at the inflow, showed identical results. The time step was set to  $\delta \tau=0.001$ , which is below the admissible value imposed by the stability requirement of the numerical scheme. Time step refinement tests showed negligible differences in the results obtained for lower  $\delta \tau$ .

# 4.2. Post-processing and comparison procedure

The numerically generated vortex ring forms rapidly, due to very small piston stroke ratios. During the injection phase, a negative vorticity layer is generated at the lateral wall, with intensity depending on the confinement. After the injection stops  $(\tau > \tau_{\text{off}})$ , a stopping vortex forms at the inlet section. To properly separate the vortex ring both from the vorticity layer at the wall and from the stopping vortex, we designed a special post-processing program, based on the free finite element solver FreeFem++ (see Hecht et al. (2007); Hecht (2012)). The velocity field obtained from the DNS simulations using second order finite-difference schemes is easily represented as a finite-element field by splitting the rectangular finite-difference cell into two triangles. The DNS values at the computational nodes are then used to completely represent the velocity field as a  $P_1$  (piece-wise linear) function on the triangular mesh. This procedure avoids interpolations between the DNS field and its representation on the finite-element grid. It is also consistent with the second order spatial accuracy of the Navier-Stokes solver. The post-processing program then identifies all closed contours for the prescribed threshold of  $\omega/\omega_{max}$ , selects the unique contour around the vortex ring centre to define the vortex ring domain  $\mathcal{D}_{VR}$  and then meshes this new domain with triangles. The interpolation between different finite-element meshes is also  $P_1$ , preserving the order of the accuracy of the finite-element representation. Since the initial finite-difference mesh is very refined, the resulting interpolation errors have no impact upon the integral values computed in the post-processing part. The advantage of this finite element post-processing is that it enables us to represent accurately and smoothly the boundary of the vortex core and then to use high-order (up to seventh order) Gauss methods to compute the integral characteristics (circulation, impulse, energy). For example, flow circulation and energy are computed as

$$\Gamma = \int \int_{\mathcal{D}_{VR}} \omega \, dr dx, \quad E = \pi \int \int_{\mathcal{D}_{VR}} \omega \Psi \, dr dx.$$
 (4.7)

The time evolution of the circulation  $\Gamma$  defined by (4.7) is shown in figure 4 (a) for a confinement parameter  $D_w/D=3$ , with  $D_w$  being the diameter of the tube. Figures 4 (b, c) illustrate the vortex ring domain  $\mathcal{D}_{VR}$  computed with the post-processing program as the contour for which  $\omega/\omega_{max}=0.05$ .

After the separation of the vortex ring domain  $\mathcal{D}_{VR}$  at a given time instant  $\tau = \tau_f$ , we can use a non-linear fit of the DNS vorticity with the VRE model (see (2.13)). Remembering that  $I_1(z) \approx \exp(z)/\sqrt{2\pi z}$  for large z, the vorticity distribution (2.13) can be approximated as:

$$\omega_{ef} = \frac{\Gamma_{ef}}{\pi} \left(\frac{R_{0f}}{r}\right)^{1/2} \frac{1}{2\beta L_{ef}^2} \exp\left(-\frac{1}{2L_{ef}^2} (r - R_{0f})^2 - \frac{1}{2(\beta L_{ef})^2} (x - X_{0f})^2\right). \tag{4.8}$$

Since FreeFem++ is interfaced to the state-of-the-art optimizer Ipopt, using the interior

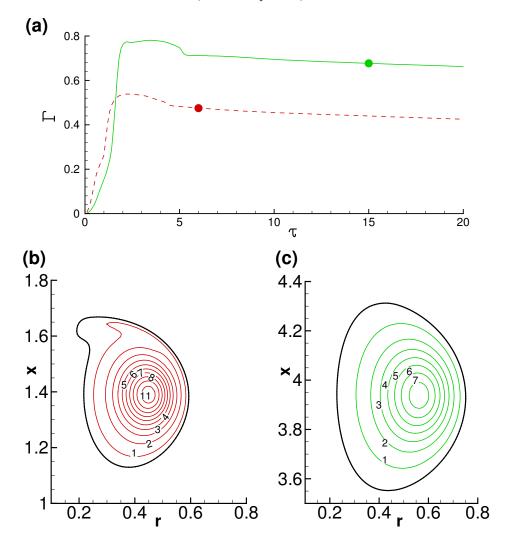


FIGURE 4. Results of the direct numerical simulation of a confined vortex ring for  $D_w/D=3$  and  $Re_{\rm D}=3400$ . (a) Time evolution of the vortex ring circulation for the cases described in table 1: Case 1  $(L_p/D=1.13, {\rm solid\ curve})$  and Case 2  $(L_p/D=0.575, {\rm dashed\ curve})$ . Contours of vorticity  $\omega/(U_0/D)$  corresponding to: (b) Case 2,  $\tau=6$  and (c) Case 1,  $\tau=15$ . In (b) and (c) the boundaries of the vortex ring domain  $\mathcal{D}_{\rm VR}$ , defined as the contour of  $\omega=0.05\,\omega_{max}$ , are plotted as thick black contours.

point minimization method suggested by Wächter (2002) and Wächter & Biegler (2006), we use a non-linear fit procedure inside the finite-element post-processing to compute the five independent parameters in (4.8):  $\Gamma_{ef}$ ,  $R_{0f}$ ,  $X_{0f}$ ,  $\frac{1}{2L_{ef}^2}$ ,  $\frac{1}{2\beta^2L_{ef}^2}$ . We finally compute  $\theta_{ef} = R_{0f}/L_{ef}$  to complete the definition of the corresponding theoretical vortex ring. Formula (2.13) is then used to reconstruct the theoretical vorticity field  $\omega(r,x)$  and compute integral characteristics in the same way as in DNS (taking into account only the vorticity of the vortex core for which  $\omega/\omega_{max} > 0.05$ ).

(a)			
Case 1 $\tau_f = 15$		g parameters $\theta_e \mid \theta_e \text{ or } \theta_c \mid \beta \mid \lambda$	$= \frac{\theta_e}{\theta_c} \left  \begin{array}{c} \text{computed integrals} \\ \Gamma_f \end{array} \right  E_f$
DNS data VRE model (elliptic) VRC model (circular)	$ \begin{vmatrix} 0.565 & 3.935 & - \\ 0.551 & 3.937 & 0.1069 \\ 0.556 & 3.937 & 0.1253 \end{vmatrix} $	5.1536   1.339   4.4383   -	- 0.6755 0.1387 1.16 0.6912 0.1447 - 0.6937 0.1452

Case 2 $\tau_f = 6$	$R_{0f} \mid X_{0f}$	ortex ring $L_e$ or $L_c$	paramet $\theta_e$ or $\theta_c$	$\beta$	$\lambda = \frac{\theta_e}{\theta_c}$	comput $\Gamma_f$	ed integrals $E_f$
DNS data VRE model (elliptic) VRC model (circular)	0.445   1.385 0.443   1.389 0.445   1.389	- 0.0754 0.0856	5.878 5.204	- 1.274 -	- 1.30 -	0.4537 0.4500 0.4494	0.0568 0.0569 0.0567

(b)

Table 2. Parameters describing the geometry of the vortex ring used in models VRE (elliptic) and VRC (circular) for run cases described in table 1. Vortex ring parameters  $(R_{0f}, X_{0f}, L, \theta)$ are computed by a non-linear fit of (4.8) using DNS data. Integral quantities (circulation  $\Gamma_f$ and energy  $E_f$ ) are then computed from models by reconstructing the vorticity field (equations (2.10) and (2.13)) and applying the same post-processing procedure as for the DNS fields (see section 4.2).  $Re_D = 3400, D_w/D = 3.$ 

### 4.3. Assessment of the new model

Figure 4 shows the time evolution of the circulation  $\Gamma$  for the two cases shown in table 1. We focus on the case of the confined vortex ring with  $Re_D = 3400$  and  $D_w/D = 3$ , for which we expect an improvement in theoretical predictions with respect to the VRC model presented in Danaila et al. (2015). After rapid increase during the injection phase, the circulation attains a small plateau and then starts to decrease during the postformation phase. The decrease rate of the circulation induced by the presence of the lateral wall was investigated experimentally by Stewart et al. (2012) and numerically by Danaila et al. (2015). For the comparison between the vortex ring models, we selected time instants in the post-formation phase of the vortex ring:  $\tau = 15$  for Case 1 (the same was used by Danaila et al. (2015) to test the VRC model) and  $\tau = 6$  for Case 2. The corresponding vortex ring vorticity contours are shown in figure 4. The results of the fit with both models VRE (elliptic core) and VRC (circular core) are summarised in table

As expected, Case 2 produces a vortex ring which is not very elongated (see figure 4 (b), but has large vorticity. As can be seen from table 2 (b), the values of the circulation  $\Gamma_f$  and energy  $E_f$  computed from (4.7) are very close to DNS values, for both VRE and VRC models. A more detailed comparison of the models is offered by the plots of the vorticity and stream function distributions. Figures 5 (a, c) show that the VRE model allows more accurate description of the vorticity distribution of the vortex core than the VRC model. At the same time, the prediction of the stream function distribution is only slightly improved in the VRE model, confirming the hypothesis that the stream function is not sensitive to the details of the vorticity distribution (see figures 5 (b, d)).

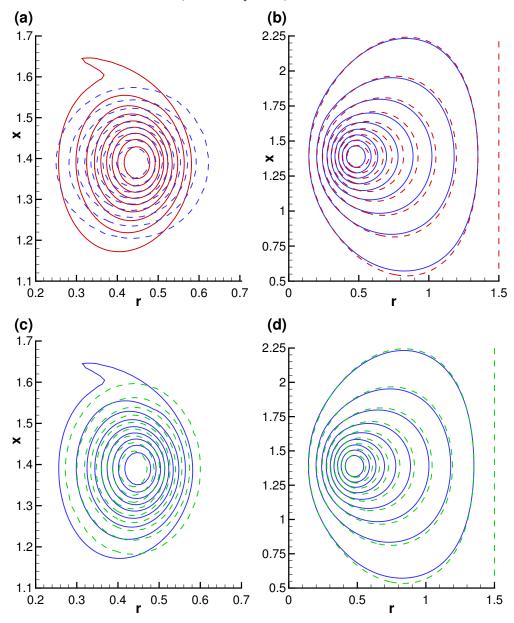


FIGURE 5. Contours of normalised vorticity  $\omega/\omega_{max}$  (a, c) and corresponding normalised stream function  $\psi/\psi_{max}$  (b, d), predicted by DNS (solid curves), the VRC (circular) model (a, b) and VRE (elliptic) model (c, d) (dashed curves). The values of  $\omega/\omega_{max}$  and  $\psi/\psi_{max}$  are shown from 0.1 to 0.9 with increments of 0.1. Case 2,  $L_p/D=0.575$ ,  $Re_{\rm D}=3400$ ,  $D_w/D=3$ ,  $\tau=6$ .

The vortex ring in Case 1 (see figure 4 (c)) is more elongated ( $\beta \approx 1.34$ ) than in Case 2. Figure 6 shows that the fit with the VRE model offers a more accurate description of both vorticity and stream function distributions, when compared to the VRC model. The integral quantities in table 2 display similar values for  $\tau = 15$ . Note that the position of the vortex centre  $(R_{0f}, X_{0f})$  and the circulation  $\Gamma_{ef}$  needed in the model (see equation 2.13) are not taken from DNS, but computed using the global fitting procedure. This

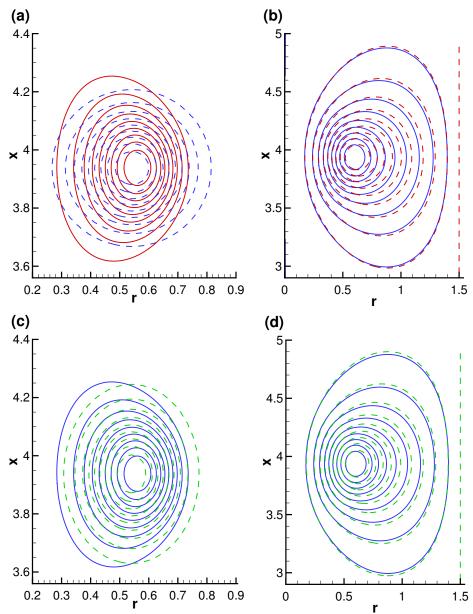


FIGURE 6. Contours of normalised vorticity  $\omega/\omega_{max}$  (a, c) and corresponding normalised stream function  $\psi/\psi_{max}$  (b, d), predicted by DNS (solid curves), the VRC (circular) model (a, b) and VRE (elliptic) model (c, d) (dashed curves). The values of  $\omega/\omega_{max}$  and  $\psi/\psi_{max}$  are shown from 0.1 to 0.9 with increments of 0.1. Case 1,  $L_p/D=1.13$ ,  $Re_{\rm D}=3400$ ,  $D_w/D=3$ ,  $\tau=15$ .

explains why, in figures 5 and 6, the centres of the fitted vortices are slightly shifted when compared to the DNS values. The elliptic shape assumed in our model approximates reasonably accurately the geometry of the DNS vortex ring, but this approximation is not perfect. The actual shape of vortex rings deviates from the theoretical elliptic one.

A different fitting procedure was also tested: the values for  $(R_{0f}, X_{0f}, \Gamma_{ef})$  were taken from DNS and the remaining parameters were fitted by a non-linear procedure, one

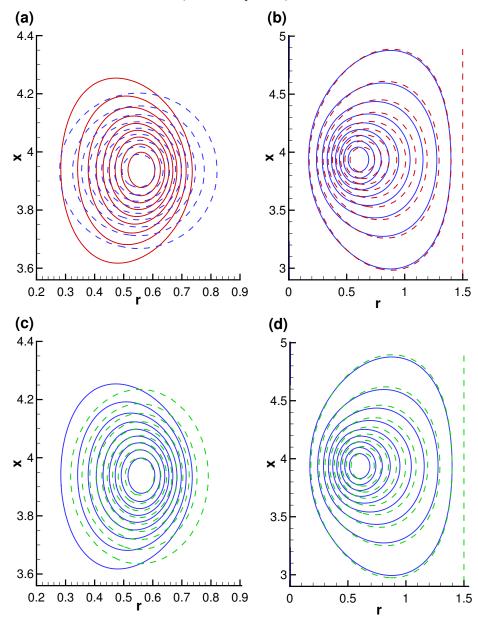


FIGURE 7. Same legend as in figure 6. Results obtained using a partial fitting procedure for the parameters of the models:  $(R_{0f}, X_{0f}, \Gamma_{ef})$  are taken from DNS and only one parameter  $(L_c)$  for the circular VRC model and two parameters  $(L_{ef}$  and  $\beta)$  for the elliptic VRE model fitted by a non-linear procedure.

parameter  $(L_c)$  for the circular VRC model and two parameters  $(L_{ef}$  and  $\beta)$  for the elliptic VRE model. This new fitting procedure has the advantage of being very fast and more convenient for a practical application of the model to describe actual, experimental or numerically generated vortex rings. It can also be useful for the reconstruction of the velocity field generated around a vortex ring when only incomplete measurements of the velocity are available (see Zhang & Danaila (2012); Danaila & Protas (2015)).

Figure 7 illustrates the obtained results for Case 1: the centre of the vortex is more accurately described with this partial fitting procedure, but only for the elliptic vortex model. The obtained values for the geometric parameters are very similar (compared to values displayed in table 2):  $\theta_c = 4.556$  (for the VRC model) and  $\theta_e = 5.19$ ,  $\beta = 1.28$  (for the VRE model).

To push the comparison procedure further and to compare the two fitting procedures, we predicted the time evolution of the energy E and translational velocity U, using the power law (2.7) for both  $\theta_c$  and  $\theta_e$ . More specifically, after fitting the DNS vortex with the prediction of the model at  $\tau = \tau_f$ , we assumed that the model parameter depends on time as:

$$\theta_e(\tau) = (\theta_e)_f \left(\frac{\tau_f}{\tau}\right)^{1/2}, \quad \beta = \text{const}, \quad \tau \geqslant \tau_f.$$
 (4.9)

Vorticity and stream function fields were then predicted for each time instant and the energy was computed from (4.7). The values of the translational velocity predicted by the model were computed from the equation (see Helmholtz (1867); Lamb (1932)):

$$U_{\text{VRE}} = \frac{R_{0f}}{\Gamma_f} \int_{-\infty}^{\infty} \int_{0}^{R_w} \left( \Psi_{\text{VRE}}^w - 6x \frac{\partial \Psi_{\text{VRE}}^w}{\partial x} \right) \omega_{\text{VRE}} dr dx / \left( 2 \int_{-\infty}^{\infty} \int_{0}^{R_w} r^2 \omega_{\text{VRE}} dr dx \right). \tag{4.10}$$

The non-dimensional translational velocity of the vortex ring was computed from DNS data as:

$$U_{\rm DNS} = \frac{dX_{\omega}}{d\tau}, \qquad X_{\omega} = \int_{-\infty}^{\infty} \int_{0}^{R_{w}} r^{2}x\omega \, dx dr / \int_{-\infty}^{\infty} \int_{0}^{R_{w}} r^{2}\omega \, dx dr, \tag{4.11}$$

where  $X_{\omega}$  is the streamwise coordinate of the vorticity centroid (see Saffman (1970)).

We display in figure 8 the model predictions for the time evolution of the energy E and translational velocity U for  $D_w/D=3$ . For this case, the influence of the boundary layer generated at the lateral wall is weak during the post-formation phase, which is in agreement with the hypothesis of the model. As a consequence, the translational velocity predicted by the VRE is closer to DNS results than the prediction of the VRC model. This result is consistent with the results of Kaplanski et al. (2012) showing that the elliptic vortex ring model allows for better predictions of the translational velocity of the unbounded vortex ring, when compared to previously developed models (Fukumoto & Moffatt 2008).

Figure 8 also compares the accuracy of the model predictions when different fitting procedures are used. As one would expect, the full fitting procedure (Figures 8 a, b) gives a better approximation of the time evolution of both energy and translational velocity. However, the partial fitting procedure (which is faster and very easy to implement) offers a fairly good approximation (Figures 8 c, d), suggesting that the model is sufficiently robust for the description of actual vortex ring flows.

Figures 6 and 8 show that the new model offers a more accurate description of the geometry of the vortex core and a more accurate prediction of the translational velocity. This is expected to be particularly important for practical applications, including those in gasoline internal combustion engines where vortex ring type flow is used to form a kernel for the ignition of a combustible mixture (Heywood 1988).

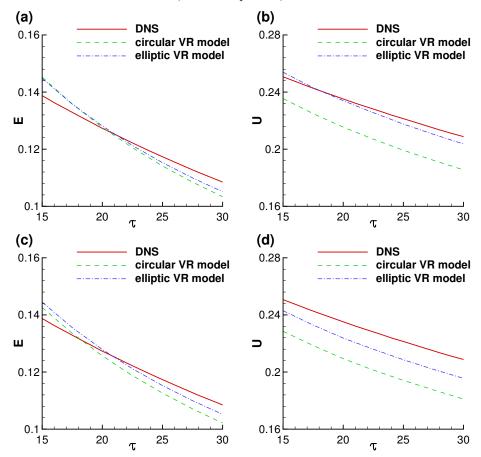


FIGURE 8. Time evolution of energy E (a) and translational velocity U (b) of the vortex ring as computed by DNS (solid curve) and predicted by models: VRC (circular) model (dashed curve) and VRE (elliptic) model (dashed-dotted curve). Case 1,  $L_p/D=1.13$ ,  $Re_{\rm D}=3400$ ,  $D_w/D=3$ . Vortex ring geometric parameters are fitted from DNS only at time instant  $\tau=15$ : (a, b) full fitting procedure (fit of 4 parameters for the VRC model and 5 parameters for the VRE model); (c, d) partial fitting procedure (fit of 1 parameter for the VRC model and 2 parameters for the VRE model).

## 4.4. Vortex ring geometry and the effect of the confinement

The new VRE model was applied to describe the vortex ring geometry during the post-formation phase. The results for Cases 1 and 2 turned out to be qualitatively similar; we show here only the results referring to Case 1. Figure 9 (a) shows that parameters  $\theta_c$ ,  $\theta_e$ , describing viscous effects (see equation 2.14), slowly decrease during the post-formation phase, indicating that the vortex ring grows by viscous dissipation. The elongation of the vortex ring, quantified by parameter  $\beta$ , increases over time, while the compression  $\lambda$  is quasi-constant during a long time period (see figure 9 (b)).

The influence of the confinement ratio  $D_w/D$  on the time evolution of parameters  $\beta$  and  $\lambda$  is shown in figure 10. Note that the evolution of these parameters reflects the interaction of the vortex ring with the lateral wall. For high confinements  $(D_w/D=2)$  the vortex ring continuously elongates and  $\beta$  increases quasi-linearly. The rate of increase of  $\beta$  diminishes for lower confinements  $(D_w/D=3)$ . Note that in predicting the time evolution of the energy and translational velocity in figure 8 we assumed that  $\beta$  was constant in time (see

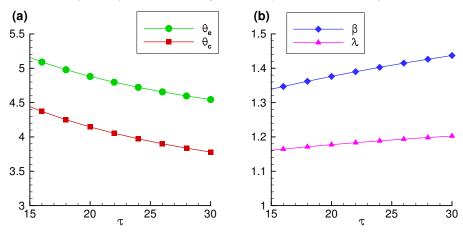


FIGURE 9. Time evolution of the parameters describing the vortex ring geometry during the post-formation phase. Parameters  $\theta_c$ ,  $\theta_e$  are related to the viscous length of the model (2.14), while  $\beta>0$  and  $\lambda>0$  describe the elongation and compression along axes x and r, respectively. All parameters are fitted from DNS data for Case 1,  $L_p/D=1.13$ ,  $Re_D=3400$ ,  $D_w/D=3$ .

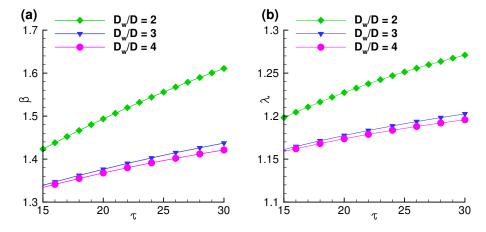


FIGURE 10. Effect of confinement  $D_w/D$  on time evolution of parameters  $\beta>0$  and  $\lambda>0$  describing the extension and compression along axes x and r, respectively. All parameters are fitted from DNS data for Case 1,  $L_p/D=1.13$ ,  $Re_{\rm D}=3400$ .

(4.9)), to facilitate the application of the model for actual vortex rings. Calculations with  $\beta(\tau)$  represented by heuristic power-laws (see also Kaplanski *et al.* (2012)) can result in slightly improved values for the predicted integral characteristics. Lower variation rates are observed in figure 10 (b) for the compression coefficient  $\lambda$ . The influence of the confinement on  $\beta$  and  $\lambda$  can be considered negligible for  $D_w/D \geqslant 3$ . This observation is in agreement with the hypothesis of the asymptotic analysis presented in section 3.1. It is also interesting to note from Figure 10 (a) that the estimate  $\beta = 1.4$  inferred from the asymptotic analysis is compatible with the values of  $\beta$  fitted from DNS data for  $D_w/D \geqslant 3$ .

## 5. An heuristic relation between vortex ring energy and circulation

In this section, we will show that the model developed above can be applied in an area not directly related to vortex ring confinements.

Recently, Krieg & Mohseni (2013) demonstrated that jet flows created from cylinderpiston mechanisms enter the reservoir with nearly parallel streamlines, whereas jets ejected through a circular orifice in a flat plate have a radial component of velocity, directed towards the axis. These authors pointed out that the converging radial velocity significantly increases the translational velocity of the produced vortex ring and proposed the well tested heuristic relation between its dimensionless energy  $E_d = E/(M^{1/2}\Gamma^{3/2})$ and circulation  $\Gamma_d = \Gamma/(M^{1/3}U^{2/3})$ , where M is the impulse:

$$\Gamma_d^{3/2} = c_1 + c_2 / E_d, \tag{5.1}$$

for a wide range of experimental results, where  $c_1 = 1.13$  and  $c_2 = 0.52$ . This relation provides a reliable tool to describe vortex rings at the initial stage of their development, when the identification of the pinch-off moment is difficult both numerically and experimentally. It allows us to approximate the translational velocity of the vortex ring (U) as a function of the total circulation  $(\Gamma)$ , impulse (M), and kinetic energy (E) of the vortex system by the following relation:

$$U = \frac{\Gamma^{3/2}E}{c_1 E M^{1/2} + c_2 M \Gamma^{3/2}}. (5.2)$$

Krieg & Mohseni (2013) showed that the Norbury family of vortex rings (Norbury 1973) satisfy (5.1) for a wide range of dimensionless core radii  $\alpha$ .

Figure 11 shows the plot of  $\Gamma_d^{3/2}$  versus  $1/E_d$ , for different vortex rings modelled with the present approach (confined elliptic, unconfined circular and unconfined elliptic). The values of  $\Gamma_d$  and  $E_d$  correspond to the range of  $L=R_0/\theta_e=0.0952381-0.285714$  and  $\theta_e=3.5-10.5$  ( $R_0=1$ ). Our aim is to find a set of geometric parameters for the vortex ring in order to obtain a linear variation of  $\Gamma_d^{3/2}$  versus  $1/E_d$ , as predicted by (5.1). As can be seen in this figure, the model with a circular core cannot be used to describe such vortex rings, while the elliptic vortex model could potentially model such flows. For both confined and unconfined elliptic vortices, the values of the geometrical parameters fitted to obtain curves describing the linear variation are indicated in the figure. We consider this to be an important generalisation of the above-mentioned result presented by Krieg & Mohseni (2013) as our model is expected to describe the geometry of vortex rings more accurately than does the Norbury family of vortex rings. This further supports the importance of the new model described in our paper.

## 6. Summary and discussion

We suggested a new model that takes into account the elongated (elliptical) shape of confined vortex rings. The vorticity  $\omega_{\text{VRE}}$  and stream function  $\Psi_{\text{VRE}}$  distributions based on the unconfined vortex ring model suggested in Kaplanski *et al.* (2012) were used in the analysis. Using Brasseur's method, a wall-induced stream function correction  $\Psi^0_{\text{VRE}}$  was analytically derived. The total stream function of the flow  $\Psi^w_{\text{VRE}}$  was presented as  $\Psi^w_{\text{VRE}} = \Psi_{\text{VRE}} - \Psi^0_{\text{VRE}}$ .

The new model was shown to describe accurately the geometry of the confined vortex

The new model was shown to describe accurately the geometry of the confined vortex ring and the time evolution of the vortex ring energy when compared with the results of Direct Numerical Simulations (DNS). For large deformations of the vortex core, occurring for high confinements, the elliptical core vortex ring model allows us to better predict the

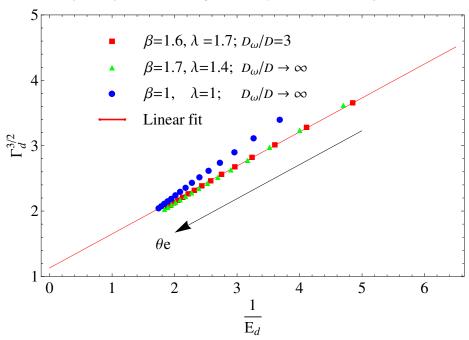


FIGURE 11. The plot of  $\Gamma_d^{3/2}$  versus  $1/E_d$ , predicted by (5.1) (Linear fit), and a set of values of  $\Gamma_d^{3/2}$  and  $1/E_d$  predicted by our model for various  $\beta$ ,  $\lambda$  and  $D_w/D$ 

time evolution of the translational velocity than the circular vortex core model suggested by Danaila *et al.* (2015). This result was expected, remembering that the model derived by Kaplanski *et al.* (2012) for an unconfined vortex ring with elliptical core offered a better estimation of the translational velocity at the whole post-formation stage when compared with experimental data presented by Weigand & Gharib (1997).

It is suggested that our model can be used to describe the deformation of the vortex ring during the post-formation phase. For long time evolution, we derived a generalisation of the well-known Rott & Cantwell (1993) formula for the drift velocity of the vortex ring by taking into account the influence of the ellipticity parameter. When compared to experimental observations, it allows us to extract the value of the ellipticity parameter  $\beta$ and thus to describe actual vortex rings with the present model. The fit of numerically or experimentally generated vortex rings with this model gives a synthetic view of the vortex elongation and compression along the longitudinal and radial axis. Two fitting procedures are suggested, computing all parameters (full fitting) or only geometrical parameters (partial fitting). The latter approach is very fast and easy to implement which makes it appropriate for practical applications involving the description of actual vortex rings or the reconstruction of their velocity field from incomplete measurements. Since the model is shown to be robust with respect to the fitting procedure, it is expected to be useful in practical applications where an accurate estimation of the vortex ring positions and geometry are required (e. q. applications in gasoline internal combustion engines, see Begg et al. (2009)). It is shown that the predictions of the model support the recently suggested heuristic relation (Krieg & Mohseni 2013) between the energy and circulation of vortex rings in the case, where the injection velocity has a converging radial component. This is believed to be an interesting feature of the model and potentially useful for practical applications.

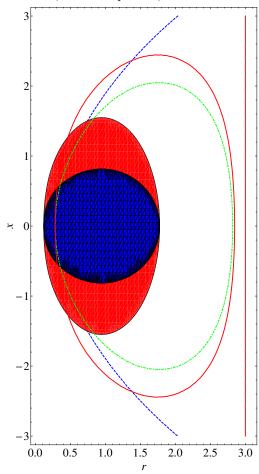


FIGURE 12. An illustration of the models for confined and unconfined vortex rings with elliptical and circular cores. Isocontours of the stream function  $\Psi/\Psi_{max}=0;0.1$  for  $\beta=1.9,D_w=3$  (solid curve),  $\beta=1,D_w/D=3$  (dashed curve) and  $\beta=1,D_w/D\to\infty$ (dot-dashed curve) with  $R_0=1,\theta=3$ . The regions of vorticity  $\omega/\omega_{max}\geqslant 0.05$  for vortex rings with elliptical (horizontal stripes) and circular (vertical stripes) cores.

The analysis presented in this paper was focused on elongation parameters  $\beta > 1$  (i. e.  $\epsilon_0 > 0$  in (2.15)) and confinement parameter  $1 < D_w/D \le \infty$ , but the new model could be regarded as a unified theoretical framework for the following models: the VRE confined elliptic vortex ring ( $\beta > 1$ ), the VRC confined circular vortex ring ( $\beta = 1$ ), and the unconfined elliptic vortex ring (when  $D_w/D \to \infty$ , correction (3.34) tends to zero). Figure 12 illustrates three possible vortex core geometries obtained for typical values of parameters  $\beta$  and  $D_w/D$ .

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# Appendix A. Proof of (3.30) for arbitrary p

In what follows the following identity will be proven:

$$\int_0^\infty \mu^{2p} \exp(-|\tilde{x}|\mu) J_0(\tilde{\tilde{r}}\mu) d\mu = \frac{2(-1)^p}{\pi} \int_0^\infty \mu^{2p} K_0(\tilde{\tilde{r}}\mu) \cos(\mu \tilde{\tilde{x}}) d\mu. \tag{A 1}$$

The left and right parts of (A1) can be calculated using Mathematica (2007) and presented in closed forms as:

$$I_1 = |\tilde{x}|^{-1-2p} \Gamma(1+2p) {}_{2}F_1(1/2+p, 1+p; 1; -\frac{\tilde{r}^2}{\tilde{x}^2}), \tag{A 2}$$

$$I_2 = \frac{(-1)^p 2^{2p}}{\pi} \tilde{r}^{-1-2p} \Gamma(1/2+p)^2 {}_2F_1(1/2+p,1/2+p;1/2;-\frac{\tilde{x}^2}{\tilde{r}^2}), \tag{A 3}$$

where

$$_{2}F_{1}\left(a,b;c;z\right) = \sum_{k=0}^{\infty} \frac{(a)_{k}(b)_{k}}{(c)_{k}} \frac{z^{k}}{k!}$$
 (A4)

is the generalised hypergeometric function and  $\Gamma(x)$  is the Gamma function. The generalised hypergeometric function is presented as (Mathematica (2007)):

$${}_{2}F_{1}(a,b;c;z) = \frac{\Gamma(b-a)\Gamma(c)}{\Gamma(b)\Gamma(c-a)} {}_{2}F_{1}\left(a,a-c+1;a-b+1;\frac{1}{z}\right) + \frac{\Gamma(a-b)\Gamma(c)}{\Gamma(a)\Gamma(c-b)} {}_{2}F_{1}\left(b,b-c+1;-a+b+1;\frac{1}{z}\right). \tag{A}5$$

This allows us to present  $I_1$  (see (A2)) in the following form

$$I_1 = |\tilde{x}|^{-1-2p} \Gamma(1+2p) \left[ \left( \frac{\tilde{r}^2}{\tilde{x}^2} \right)^{-1-p} \frac{\Gamma(-1/2)\Gamma(1)}{\Gamma(-p)\Gamma(1/2+p)} {}_2F_1 \left( 1+p, 1+p; 3/2; -\frac{\tilde{x}^2}{\tilde{r}^2} \right) \right]$$

$$+ \left(\frac{\tilde{r}^2}{\tilde{x}^2}\right)^{-1/2-p} \frac{\Gamma(1/2)\Gamma(1)}{\Gamma(1/2-p)\Gamma(1+p)} {}_2F_1\left(1/2+p,1/2+p;1/2;-\frac{\tilde{x}^2}{\tilde{r}^2}\right) ] \tag{A 6}$$

Taking into account that  $\Gamma(-p) = \tilde{\infty}$ ;  $p \in N$  ( $\tilde{\infty}$  is complex infinity (Mathematica (2007))), we can ignore the first term in the square brackets. Remembering that  $\Gamma(1) = 1$ , and  $\Gamma(1/2) = \sqrt{\pi}$  we obtain, after tedious but straightforward calculations

$$I_1 = \frac{\sqrt{\pi}\Gamma(1+2p)}{\Gamma(1/2-p)\Gamma(1+p)}\tilde{r}^{-1-2p}{}_2F_1\left(1/2+p,1/2+p;1/2;-\frac{\tilde{x}^2}{\tilde{r}^2}\right). \tag{A 7}$$

Remembering that  $\Gamma(z)\Gamma(z+1/2)=\sqrt{\pi}\,2^{1-2z}\Gamma(2z)$  for z=p+1/2 (see Abramowitz & Stegun (1964)) we can write:

$$\frac{\Gamma(1+2p)}{\Gamma(1+p)} = \frac{2^{2p}\Gamma(1/2+p)}{\sqrt{\pi}}.$$
 (A8)

Using the relations

$$\frac{1}{\Gamma(1/2-p)} = \frac{(2p-1)!!}{(-2)^p \sqrt{\pi}}, \quad \Gamma(1/2+p) = \sqrt{\pi} \frac{(2p-1)!!}{2^p}, \tag{A 9}$$

we can estimate  $I_1$  as

$$I_1 = \frac{(-1)^p (2)^{2p}}{\pi} \Gamma(1/2 + p)^2 \tilde{r}^{-1-2p} {}_2 F_1 \left( 1/2 + p, 1/2 + p; 1/2; -\frac{\tilde{x}^2}{\tilde{r}^2} \right). \tag{A 10}$$

Thus, integral  $I_1$  is identical to  $I_2$  and (3.30) is proven.

#### REFERENCES

- ABRAMOWITZ, M. & STEGUN, I. A. 1964 Handbook of Mathematical Functions. New York: Dover Publications.
- Batchelor, G. K. 1988 An Introduction to Fluid Dynamics, 7th edn. Cambridge, New York: Cambridge University Press.
- Begg, S., Kaplanski, F., Sazhin, S., Hindle, M. & Heikal, M. 2009 Vortex ring-like structures in gasoline fuel sprays under cold-start conditions. *International Journal of Engine Research* **10** (4), 195–214.
- Brasseur, J. G. 1979 Kinematics and dynamics of vortex rings in a tube. Ph.D. Thesis, report JIAA, TR-26, Joint Institute for Aeronautics and Acoustics, Dept. of Aeronautics and Astronautics, Stanford University.
- Brasseur, J. G. 1986 Evolution characteristics of vortex rings over a wide range of Reynolds numbers. AIAA Paper pp. 1097:1–11.
- CANDON, S. & MARSHALL, J. 2012 Vortex ring deformation, capture, and entrainment by a columnar vortex. Phys. Fluids 24, 093604.
- Cater, J.E., Soria, J. & Lim, T.T. 2004 The interaction of the piston vortex with a piston-generated vortex ring. *J. Fluid Mech.* 499, 327–343.
- Dabiri, J. O. & Gharib, M. 2005 Starting flow through nozzles with temporally variable exit diameter. J. Fluid Mech. 538, 111–136.
- Danaila, I. & Helie, J. 2008 Numerical simulation of the postformation evolution of a laminar vortex ring. *Phys. Fluids* **20**, 073602.
- Danaila, I., Kaplanski, F. & Sazhin, S. 2015 Modelling of confined vortex rings. *J. Fluid Mech.* 774, 267–297.
- Danaila, I. & Protas, B. 2015 Optimal reconstruction of inviscid vortices. Proceedings of the Royal Society A: Mathematical, Physical & Engineering Sciences 471, 20150323.
- Danaila, I., Vadean, C. & Danaila, S. 2009 Specified discharge velocity models for the numerical simulation of laminar vortex rings. *Theor. Comput. Fluid Dynamics* **23**, 317–332.
- Fukumoto, Y. 2010 Global evolution of viscous vortex rings. *Theor. Comput. Fluid Dynamics* **24**, 335–347.
- Fukumoto, Y. & Moffatt, H. K. 2000 Motion and expansion of a viscous vortex ring. Part 1. A higher-order asymptotic formula for the velocity. *J. Fluid Mech.* **417**, 1–45.
- Fukumoto, Y. & Moffatt, H. K. 2008 Kinematic variational principle for motion of vortex rings. *Physica D* **237**, 2210–2217.
- GHARIB, M., RAMBOD, E., KHERADVAR, A., SAHN, D. J. & DABIRI, J. O. 2006 Optimal vortex formation as an index of cardiac health. *Proceedings of the National Academy of Sciences* (PNAS) 103, 6305–6308.
- GHARIB, M., RAMBOD, E. & SHARIFF, K. 1998 A universal time scale for vortex ring formation. J. Fluid Mech. 360, 121–140.
- HECHT, F. 2012 New developments in Freefem++. Journal of Numerical Mathematics 20, 251–266.
- HECHT, F., PIRONNEAU, O., HYARIC, A. LE & OHTSUKE, K. 2007 FreeFem++ (manual). www.freefem.org.
- HELMHOLTZ, H. 1867 On integrals of the hydrodynamical equations, which express vortex-motion. *Phil. Mag. (ser. 4)* **33**, 485–510.
- HEYWOOD, J. B. 1988 Internal Combustion Engines Fundamentals. New York: McGraw-Hill.
- Kaplanski, F., Sazhin, S. S., Fukumoto, Y., Begg, S. & Heikal, M. 2009 A generalized vortex ring model. J. Fluid Mech. 622, 233–258.
- Kaplanski, F. B., Fukumoto, Y. & Rudi, Y. A. 2012 Reynolds-number effect on vortex ring evolution in a viscous fluid. *Phys. Fluids* **24**, 033101.
- KAPLANSKI, F. B. & RUDI, Y. A. 1999 Dynamics of a viscous vortex ring. Int. J. Fluid Mech. Res. 26, 618–630.

- KAPLANSKI, F. B. & RUDI, Y. A. 2005 A model for the formation of 'optimal' vortex ring taking into account viscosity. Phys. Fluids 17, 087101–087107.
- KRIEG, M. & MOHSENI, K. 2013 On the approximation of the translational velocity of vortex rings. J. Fluids Engineering 135, 124501.
- KRUEGER, S. & GHARIB, M. 2003 The significance of vortex ring formation on the impulse and thrust of a starting jet. *Phys. Fluids* **15**, 1271.
- Lamb, H. 1932 Hydrodynamics. Dover, New York.
- LIM, T. T. & NICKELS, T. B. 1995 *Vortex rings*, in Vortices in Fluid Flows, edited by S. I. Green, Kluwer, Dordrecht, 1995, p. 95.
- MATHEMATICA 2007 MATHEMATICA Book. version 6.0.0, http://functions.wolfram.com Wolfram Research Inc.
- Norbury, J. 1973 A family of steady vortex rings. J. Fluid Mech. 57, 417–431.
- ROTT, N. & CANTWELL, B. 1993 Vortex drift. I: Dynamic interpretation. *Phys. Fluids* 5, 1443–1450.
- Saffman, P. G. 1970 The velocity of viscous vortex rings. Studies in Applied Math. 49, 371.
- Sazhin, S. S. 2014 Droplets and Sprays. London: Springer.
- Shariff, K. & Leonard, A. 1992 Vortex rings. Ann. Rev. Fluid Mech. 24, 235–279.
- Stewart, K., Niebel, C., Jung, S. & Vlachos, P. 2012 The decay of confined vortex rings. Experiments in Fluids 53, 163–171.
- WÄCHTER, A. 2002 An interior point algorithm for large-scale nonlinear optimization with applications in process engineering. Ph.D. Thesis, Carnegie Mellon University, Pittsburgh, PA, USA.
- Wächter, A. & Biegler, L. T. 2006 On the implementation of an interior-point filter linesearch algorithm for large-scale nonlinear programming. *Mathematical Programming* **106**, 25–57.
- WEIGAND, A. & GHARIB, M. 1997 On the evolution of laminar vortex rings. Exp. Fluids 22, 447–457.
- ZHANG, Y. & DANAILA, I. 2012 A finite element BFGS algorithm for the reconstruction of the flow field generated by vortex rings. *Journal of Numerical Mathematics* **20**, 325–340.
- Zhao, W., Steven, H. F. & Mongeau, L. G. 2000 Effects of trailing jet instability on vortex ring formation. *Phys. Fluids* **12**, 589–596.